22.314 Problem V Solution Fall 2006

Known

E = 195000 v = 0.3

 $\sigma A = 150 \quad \sigma B = 260 \quad \epsilon B = 0.54 \cdot 10^{-2}$

Derived properties

emA := 0

 $\epsilon mB := \epsilon B - \frac{\sigma B}{E}$

EmY = 0.002

 $\sigma Y \coloneqq \frac{\epsilon m Y - \epsilon m A}{\epsilon m B - \epsilon m A} \cdot (\sigma B - \sigma A) + \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get yield stress } \sigma Y = \sigma A \text{ Linear interpolation between A and B to get y = \sigma A \text{ Linear interpolatin between A and B to get y = \sigma A \text{ Li$

 $\sigma Y = 204.098$

The 0.2% offset yield stress oY is 204.098 MPa

From the uniaxial stress-strain curve, when $\sigma p \le \sigma A$, $\epsilon e = \epsilon e A = 0$; when $\sigma A \le \sigma p \le \sigma B$, ϵe is linear on σp ; when $\sigma p = \sigma B$, $\epsilon e = \epsilon e B = \epsilon B - \sigma B/E$

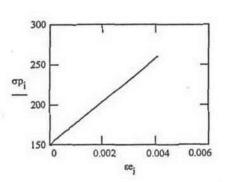
$$\varepsilon eA := 0$$

 $\varepsilon eB := \varepsilon B - \frac{\sigma B}{E}$

i = 0.. 100

 $\varepsilon e_i = \varepsilon e B \cdot \frac{i}{100}$

 $\sigma p_i := \frac{\epsilon e_i - \epsilon e A}{\epsilon e B - \epsilon e A} \cdot (\sigma B - \sigma A) + \sigma A \qquad \textit{//} \sigma p \text{ as a function of } \epsilon e$



Loading sequences and questions

i

	/137.5	- 12.5	75
σa :=	- 12.5	137.5	75
	75	75	50

 $\sigma kk := \sigma a_{0,0} + \sigma a_{1,1} + \sigma a_{2,2}$

σkk = 325 i := 0..2 j = 0..2

σae := 0

$$Sa_{i,j} := \sigma a_{i,j} - \frac{1}{3} \cdot \delta(i - j, 0) \cdot \sigma kk$$

$$Sa = \begin{pmatrix} 29.167 & -12.5 & 75 \\ -12.5 & 29.167 & 75 \\ 75 & 75 & -58.333 \end{pmatrix}$$

$$\sigma ae = \sqrt{\frac{3}{2} \cdot \left[\left(Sa_{0,0} \right)^2 + \left(Sa_{0,1} \right)^2 \cdot 2 + \left(Sa_{0,2} \right)^2 \cdot 2 + \left(Sa_{1,1} \right)^2 + \left(Sa_{1,2} \right)^2 \cdot 2 + \left(Sa_{2,2} \right)^2 \right]}$$

orae = 204.634

Now calculate VonMises stress for comparision

eigena := eigenvals(σa)

eigena =
$$\begin{pmatrix} 150\\ 200\\ -25 \end{pmatrix}$$

 $\sigma VM := \sqrt{\frac{1}{2} \cdot [(150 - 200)^2 + (150 + 25)^2 + (200 + 25)^2]}$

 $\sigma VM = 204.634$

We can see oae=oVM

2) Strain tensor for oa

a) Elastic strain

i:=0..2

j:=0..2

 $\epsilon ela_{i,j} = \frac{1}{E} \cdot \left[(1 + \nu) \cdot \sigma a_{i,j} - \nu \cdot \delta(i - j, 0) \cdot \sigma kk \right]$

	4.167.10 ⁻⁴	-8.333•10 ⁻⁵	5.10-4
εela =	-8.333•10 ⁻⁵	4.167.10 ⁻⁴	
	5-10-4	5.10-4	-1.667•10 ⁻⁴

b) Mechanical strain

Since oae=204>oa=150, mechanical strain exists and has to be calculated

Since the stress was applied in a proportional manner, Saij/σe is constant, thus we have εm(i,j)=(3*Sa(i,j)/2*σe)*εe

sae = 1

roota = root
$$\sigma ae - \left[\frac{\epsilon ae - \epsilon eA}{\epsilon eB - \epsilon eA} \cdot (\sigma B - \sigma A) + \sigma A\right], \epsilon ae$$

εae ∶= roota

 $\varepsilon ae = 0.002$

100·εae = 0.202 // Mathcad sometimes doesn't show enough significant digits. εae is actually 0.00202 here

 $\varepsilon \operatorname{ma}_{i,j} = \frac{3 \cdot \operatorname{Sa}_{i,j}}{2 \cdot \sigma \operatorname{ae}} \cdot \varepsilon \operatorname{ae}$

$$\varepsilon \text{ma} = \begin{bmatrix} 4.318 \cdot 10^{-4} & -1.851 \cdot 10^{-4} & 0.001 \\ -1.851 \cdot 10^{-4} & 4.318 \cdot 10^{-4} & 0.001 \\ 0.001 & 0.001 & -8.637 \cdot 10^{-4} \end{bmatrix}$$

Total stress

 ε tola = ε ela + ε ma

 $\varepsilon \text{tola} = \begin{pmatrix} 8.485 \cdot 10^{-4} & -2.684 \cdot 10^{-4} & 0.002 \\ -2.684 \cdot 10^{-4} & 8.485 \cdot 10^{-4} & 0.002 \\ 0.002 & 0.002 & -0.001 \end{pmatrix}$

3) When stress is proportionally reduced to zero, ϵ ela vanishs while ϵ ma remains unchanged. ϵ tola= ϵ ma when zero stress is reached

$$\sigma b := \begin{pmatrix} 260 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{pmatrix}$$

 $\sigma kk := \sigma b_{0,0} + \sigma b_{1,1} + \sigma b_{2,2}$

 $\sigma kk = 260$

j:=0..2

$$Sb_{i,j} := \sigma b_{i,j} - \frac{1}{3} \cdot \delta(i-j,0) \cdot \sigma kk$$
$$Sb = \begin{pmatrix} 173.333 & 0 & 0 \\ 0 & -86.667 & 0 \\ 0 & 0 & -86.667 \end{pmatrix}$$

$$\sigma be = \sqrt{\frac{3}{2} \cdot \left[\left(Sb_{0,0} \right)^2 + \left(Sb_{0,1} \right)^2 \cdot 2 + \left(Sb_{0,2} \right)^2 \cdot 2 + \left(Sb_{1,1} \right)^2 + \left(Sb_{1,2} \right)^2 \cdot 2 + \left(Sb_{2,2} \right)^2 \right]}$$

σbe = 260

4) Strain tensors for σb

a) Elastic strain

i := 0..2

j:=0..2

 $\epsilon elb_{i,j} = \frac{1}{E} \cdot \left[(1 + \nu) \cdot \sigma b_{i,j} - \nu \cdot \delta(i - j, 0) \cdot \sigma kk \right]$

$$\varepsilon elb = \begin{pmatrix} 0.001 & 0 & 0 \\ 0 & -4 \cdot 10^{-4} & 0 \\ 0 & 0 & -4 \cdot 10^{-4} \end{pmatrix}$$

b) Mechanical strain

During the second loading, when σe increases from 0 to σae , no mechanical strain is produced. Since $\sigma be=260 > \sigma ae=204$, new mechanical strain will be produced when σe increases from σae to σbe .

εbe := 1

rootb = root
$$\left[\sigma be - \left[\frac{\varepsilon be - \varepsilon eA}{\varepsilon eB - \varepsilon eA} \cdot (\sigma B - \sigma A) + \sigma A\right], \varepsilon be\right]$$

ebe := rootb

εbe = 0.004

100·εbe = 0.407

$$\varepsilon$$
 madd_{i,j} := $\frac{3 \cdot Sb_{i,j}}{2 \cdot \sigma be} \cdot (\varepsilon be - \varepsilon ae)$

	0.002	0	0 1
ɛmadd =	0	-0.001	0
	0	0	-0.001

 ε madd is the addtional mechanical strain tensor that is produced when σe increases from σae to σbe . Adding ε madd to ε ma we can get the total mechanical strain when σb is reached.

 ε mb := ε madd + ε ma

	0.002	$-1.851 \cdot 10^{-4}$	0.001	
εmb =	-1.851-10-4	-5.916•10 ⁻⁴	0.001	
	0.001	0.001	-0.002	

c) Total stress

 ε tolb := ε elb + ε mb

	0.004	$-1.851 \cdot 10^{-4}$	0.001
εtolb =	-1.851-10-4	-9.916 10-4	0.001
	0.001	0.001	-0.002

5) When stress is proportionally reduced to zero, εelb vanishs while εmb remains unchanged. εtolb=εmb when zero stress is reached

Note: Some of the elements in the matrices may not be accurate because Mathcad sometimes doesn't show enough significant digits.