## Problem Set No. 2

Out: Thursday, March 1, 2007
Due: Thursday, March 15, 2007 in class

## Problem 1

The cylinder rolls back and forth without slip as shown in the figure below.
(a) Show that the equation of motion can be written in the form

$$
\dot{x}+\omega^{2}\left[1-l\left(1+x^{2}\right)^{-1 / 2}\right] x=0
$$

where $\omega^{2}=2 k / 3 M$ and $l$ is the free length of the spring. All lengths were made dimensionless with respect to the radius $r$.
(b) Sketch the potential energy as a function of $x$ for
(i) $1 \leq l$
(ii) $1>l$

Show the equilibrium positions and indicate whether they are stable or unstable.
(c) For $l=\sqrt{2}$, obtain a two-term frequency-amplitude relationship for small oscillations around the equilibrium position.


## Problem 2

Consider a simple pendulum with a dashpot as shown below.
(a) Show that the equation of motions is

$$
\begin{equation*}
m l_{1} \ddot{\theta}=-m g \sin \theta-\hat{\mu} m l_{1} \dot{\theta} \cos ^{2}(\beta-\theta) . \tag{1}
\end{equation*}
$$

Then show that (1) can also be written as

$$
\begin{equation*}
\ddot{\theta}+\omega^{2} \sin \theta+\frac{\hat{\mu}\left(l_{1}+l_{2}\right)^{2} \sin ^{2} \theta}{l_{2}^{2}+2 l_{1}\left(l_{1}+l_{2}\right)(1-\cos \theta)} \dot{\theta}=0 . \tag{2}
\end{equation*}
$$


(b) Expanding and retaining through the cubic terms, show that (2) becomes

$$
\begin{equation*}
\ddot{\theta}+\omega^{2}\left(1-\frac{1}{6} \theta^{2}\right) \theta+2 \mu \theta^{2} \dot{\theta}=0 \tag{3}
\end{equation*}
$$

where

$$
\begin{equation*}
2 \mu=\frac{\hat{\mu}\left(l_{1}+l_{2}\right)^{2}}{l_{2}^{2}} \tag{4}
\end{equation*}
$$

Using (3), obtain the following first approximation for $\theta$ when the amplitude of the motion is small but finite:

$$
\begin{equation*}
\theta=\frac{a_{0}}{\sqrt{1+\frac{1}{2} \mu a_{0}^{2} t}} \cos \left\{\omega\left[t-\frac{\ln \left(1+\frac{1}{2} \mu a_{0}^{2} t\right)}{8 \mu}\right]+\beta_{0}\right\} \tag{5}
\end{equation*}
$$

where $a_{0}$ and $\beta_{0}$ are constants of integration. Note that $\mu$ is not small and that in this case the frequency is affected by the damping in the first approximation. As a check, show that in the limit as $\mu \rightarrow 0$ equation (5) reduces to

$$
\begin{equation*}
\theta=a_{0} \cos \left[\omega\left(1-\frac{1}{16} a_{0}^{2}\right) t+\beta_{0}\right] . \tag{6}
\end{equation*}
$$

## Problem 3

The response of a nonlinear system to harmonic excitation is governed by the following equation:

$$
\ddot{x}+2 \zeta \dot{x}|\dot{x}|+x+\beta \epsilon x^{3}=\cos \frac{\Omega}{\omega_{0}} t,
$$

where $\Omega / \omega_{0} \approx 1$. Assume light damping $(\zeta \ll 1)$ and weak nonlinearity $(0<\epsilon \ll 1)$ with $\beta=O(1)$.
(a) Find the appropriate scaling of the small parameter $\zeta$, in terms of $\epsilon$, so that light damping and weak nonlinearity balance. What is the width and height of the resonance peak in terms of $\epsilon$ ?
(b) Under the assumptions in (a), derive evolution equations for the response. Which method out of the three we learned in class (Poincaré-Lindstedt, multiple scales, averaging) is best suited for this problem? Why?
(c) Obtain the frequency-response equation. Is there a jump phenomenon associated with this motion? Is this motion bounded?

## Problem 4

A two-degree-of-freedom system is governed by the following coupled (dimensionless) equations

$$
\left\{\begin{array}{l}
\frac{d^{2} x}{d t^{2}}+\omega_{1}^{2} x=y \\
\frac{d^{2} y}{d t^{2}}+\omega_{2}^{2} y=\epsilon^{2} \beta x^{3}
\end{array}\right.
$$

subject to initial conditions

$$
x(0)=x_{0}, \quad y(0)=y_{0}, \quad \dot{x}(0)=u_{0}, \quad \dot{y}(0)=v_{0} .
$$

Here $\epsilon$ is a measure of nonlinearity and $\beta$ is an $O(1)$ parameter.
In the limit $\epsilon \rightarrow 0$ and away from resonance $\left(\omega_{1} \neq \omega_{2}\right)$, the linear response of this system consists of two harmonics with frequencies $\omega_{1}$ and $\omega_{2}$ :

$$
\begin{gathered}
y(t)=y_{0} \cos \omega_{2} t+\frac{v_{0}}{\omega_{2}} \sin \omega_{2} t, \\
x(t)=\frac{y(t)}{\omega_{1}^{2}-\omega_{2}^{2}}+\frac{1}{\omega_{1}}\left(u_{0}-\frac{v_{0}}{\omega_{1}^{2}-\omega_{2}^{2}}\right) \sin \omega_{1} t+\left(x_{0}-\frac{y_{0}}{\omega_{1}^{2}-\omega_{2}^{2}}\right) \cos \omega_{1} t .
\end{gathered}
$$

Note that in this expression $x(t)$ becomes unbounded at resonance $\left(\omega_{2}=\omega_{2}\right)$.
Your job is to construct a uniformly valid expansion that describes the weakly nonlinear $(0<\epsilon \ll 1)$ response of this system near resonance conditions ( $\omega_{1} \approx \omega_{2}$ ).

Clue: Based on the linear response exactly at resonance $\left(\omega_{1}=\omega_{2}\right)$, use a 'naive' expansion to deduce the timescale on which nonlinear effects come into play as well as the appropriate re-scaling of $x$ and $y$ near resonance.

