## Elasticity of Rocks

## Assigned Reading:

Gueguen, Yves and V. Palcisukas, Introduction to the Physics of Rocks, Chapter III, p. 53-61; Chapter IV. 73-92. (Essentially a discussion of some of the material we have covered over the last couple of week. I hope this reading is easier no than it would have been before we started.)+

## Resource reading:

Atkinson BK (1987) Fracture Mechanics of Rock. Academic Press, London UK, pp 534 Chapters 1\&11
Johnson AM (1970) Physical Processes in Geology. Freeman Cooper, San Francisco CA, Chapters 9-11.
Clyne, T. W., and P. J. Withers, An Introduction to Metal Matrix Composites, Cambridge University Press, 1993.
Hearmon, R. F. S., An Introduction to Applied Anisotropic Elasticity, Oxford University Press, 1961.
Mavko, G., T. Mukerji, and J. Dvorkin, The Rock Physics Handbook, Cambridge Univ. Press, 1998.
Watt, J. P., G. F. Davies, and R. J. O’Connell, "The Elastic Properties of Composite Materials", Rev. Geophys. Space Phys., 14, 541-563, 1976.
Muskhelishvili, N. I.(Nikolaĭ Ivanovich),1891-Some basic problems of the mathematical theory of elasticity; fundamental equations, plane theory of elasticity, torsion, and bending, Groningen, P. Noordhoff, 1963. QA931.M9871 1954

## Exact Elastic Treatment of Simple Geometry

Fundamental Equations of Elasticity:
Equilibrium, Compatibility, Constitutive Law
Biharmonic Equation, Airy Stress Function
Particular Solutions
Internally Pressurized Cylinder
Externally Stressed Hole
Externally Stressed Elliptical Hole
Elastic Moduli of Cracked Solids

## Basic Equations of Isotropic Elasiticity

Elasticity of isotropic elastic materials

$$
\varepsilon_{i j}=\frac{(1+v)}{E} \sigma_{i j}-\frac{v}{E} \delta_{i j} \sigma_{k k}
$$

$$
\sigma_{i j}=\lambda \delta_{i j} \varepsilon_{k k}+2 \mu \varepsilon_{i j}
$$

## Equilibrium (Static)

$$
\frac{\partial \sigma_{i j}}{\partial x_{j}}=0
$$

## Compatibility

$$
\frac{\partial^{2} \varepsilon_{i j}}{\partial x_{k} \partial x_{l}}+\frac{\partial^{2} \varepsilon_{k l}}{\partial x_{i} \partial x_{j}}-\frac{\partial^{2} \varepsilon_{i k}}{\partial x_{j} \partial x_{l}}-\frac{\partial^{2} \varepsilon_{j l}}{\partial x_{i} \partial x_{k}}=0
$$

Proof:
Strain may be viewed as the differential of the displacement change vector. Provided that no gaps open up and that no overlaps develop, the displacements must be continuous.
If the displacement change vector, $\Delta \mathbf{u}=\left[u_{1}, u_{2}, u_{3}\right]$, then

$$
\begin{aligned}
& \varepsilon_{11}=\frac{\partial u_{1}}{\partial x_{1}} \quad \text { and } \quad \frac{\partial^{2} \varepsilon_{11}}{\partial^{2} x_{2}}=\frac{\partial^{3} u_{1}}{\partial^{2} x_{2} \partial x_{1}} \\
& \text { calculate } \frac{\partial^{2} \varepsilon_{22}}{\partial^{2} x_{1}} \text { and } \frac{\partial^{2} \varepsilon_{12}}{\partial x_{1} \partial x_{2}} . \text { Notice that } \frac{\partial^{2} \varepsilon_{22}}{\partial^{2} x_{1}}+\frac{\partial^{2} \varepsilon_{11}}{\partial^{2} x_{2}}=2 \frac{\partial^{2} \varepsilon_{12}}{\partial x_{1} \partial x_{2}}
\end{aligned}
$$

In general any problem for the statics of a three dimensional body loaded externally and with no body forces is well-posed using these equations as long as sufficient boundary conditions are given.

## Consider plane strain and plane stress (i.e. 2-D) without body forces:

Equilibrium

$$
\begin{aligned}
& \frac{\partial \sigma_{11}}{\partial x_{1}}+\frac{\partial \sigma_{12}}{\partial x_{2}}=0 \\
& \frac{\partial \sigma_{21}}{\partial x_{1}}+\frac{\partial \sigma_{22}}{\partial x_{2}}=0
\end{aligned}
$$

Compatibility

$$
\frac{\partial^{2} \varepsilon_{22}}{\partial^{2} x_{1}}+\frac{\partial^{2} \varepsilon_{11}}{\partial^{2} x_{2}}=2 \frac{\partial^{2} \varepsilon_{12}}{\partial x_{1} \partial x_{2}}
$$

Elasticity

$$
\begin{aligned}
& \varepsilon_{11}=\frac{(1+v)}{E} \sigma_{11}-\frac{v}{E}\left(\sigma_{11}+\sigma_{22}\right)=\frac{\sigma_{11}}{E}-\frac{v \sigma_{22}}{E} \\
& \varepsilon_{12}=\frac{(1+v) \sigma_{12}}{E}
\end{aligned}
$$

## Equilibrium, Compatibility and Elasticity require

$$
\left(\frac{\partial^{2}}{\partial x_{1}^{2}}+\frac{\partial^{2}}{\partial x_{2}^{2}}\right)\left(\sigma_{11}+\sigma_{22}\right)=0=\nabla^{2} \sigma_{i i}
$$

## Airy Stress Function:

Assume that there exists a function, $\Phi$, the Airy stress function, such that $\frac{\partial^{2} \Phi}{\partial x_{2}{ }^{2}}=\sigma_{11}$
$\frac{\partial^{2} \Phi}{\partial x_{1}{ }^{2}}=\sigma_{22}$
$-\frac{\partial^{2} \Phi}{\partial x_{1} \partial x_{2}}=\sigma_{12}$
Then from above,
$\nabla^{4} \Phi=0 \quad$ Biharmonic Equation

Biharmonic Equation: Method of solution is 1.)Find a general Airy's stress function with appropriate
symmetry; 2.) Use differential equations to get stress;
3.) Use boundary conditions to find arbitrary
constants: 4.) Use Hook's law to get strains; 5.)
Integrate to get displacements

## Elasticity in Problems with Cylindrical and Spherical Symmetry

 In problems with cylindrical symmetry, $r, \theta, z$ :
## Strain:

$$
\begin{array}{lll}
\varepsilon_{r r}=\frac{\partial u_{r}}{\partial r}, & \varepsilon_{\theta \theta}=\frac{1}{r} \frac{\partial u_{\theta}}{\partial \theta}+\frac{u_{r}}{r}, & \varepsilon_{z z}=\frac{\partial u_{z}}{\partial z} \\
\varepsilon_{r \theta}=\frac{1}{2}\left(\frac{\partial u_{\theta}}{\partial r}+\frac{1}{r} \frac{\partial u_{r}}{\partial \theta}-\frac{u_{\theta}}{r}\right), & \varepsilon_{\theta z}=\frac{1}{2}\left(\frac{1}{r} \frac{\partial u_{z}}{\partial \theta}+\frac{\partial u_{\theta}}{\partial z}\right), & \varepsilon_{r z}=\frac{1}{2}\left(\frac{\partial u_{r}}{\partial z}+\frac{\partial u_{z}}{\partial r}\right)
\end{array}
$$

The gradient operator and the Laplacian in cylindrical coordinates are

$$
\begin{aligned}
& \nabla=\hat{\mathbf{e}}_{r} \frac{\partial}{\partial r}+\hat{\mathbf{e}}_{\theta} \frac{1}{r} \frac{\partial}{\partial \theta}+\hat{\mathbf{e}}_{z} \frac{\partial}{\partial z} \\
& \nabla^{2}=\frac{1}{r} \frac{\partial}{\partial r}\left(r \frac{\partial}{\partial r}\right)+\frac{1}{r^{2}} \frac{\partial^{2}}{\partial \theta^{2}}+\frac{\partial^{2}}{\partial z^{2}}
\end{aligned}
$$

## Equilibrium:

In problems with cylindrical symmetry, $r, \theta$, $z$ :

$$
\begin{aligned}
& \frac{\partial \sigma_{r r}}{\partial r}+\frac{1}{r} \frac{\partial \sigma_{r \theta}}{\partial \theta}+\frac{\partial \sigma_{z \theta}}{\partial z}+\frac{\sigma_{r r}-\sigma_{\theta \theta}}{r}=0 \\
& \frac{\partial \sigma_{r \theta}}{\partial r}+\frac{1}{r} \frac{\partial \sigma_{\theta \theta}}{\partial \theta}+\frac{\partial \sigma_{z \theta}}{\partial z}+2 \frac{\sigma_{r \theta}}{r}=0 \\
& \frac{\partial \sigma_{z r}}{\partial r}+\frac{1}{r} \frac{\partial \sigma_{z \theta}}{\partial \theta}+\frac{\partial \sigma_{z z}}{\partial z}+\frac{\sigma_{z r}}{r}=0
\end{aligned}
$$

## Spherical symmetry, r, $\theta$, $\phi$,

Special case that displacements occur in the radial direction only:
Strain:
In problems with spherical symmetry, $r, \theta, \phi$, for the special case that displacements occur in the radial direction only:

$$
\varepsilon_{r r}=\frac{\partial u_{r}}{\partial r}, \quad \quad \varepsilon_{\phi \phi}=\varepsilon_{\theta \theta}=\frac{u_{r}}{r}, \quad \varepsilon_{r \theta}=\varepsilon_{\theta z}=\varepsilon_{r z}=0
$$

## Equilibrium:

$$
\frac{\partial \sigma_{r r}}{\partial r}+\frac{1}{r}\left(2 \sigma_{r r}-\sigma_{\theta \theta}-\sigma_{\phi \phi}\right)=0, \quad \frac{\partial \sigma_{\theta \theta}}{\partial \theta}=\frac{\partial \sigma_{\phi \phi}}{\partial \phi}=0, \quad \sigma_{\theta \theta}=\sigma_{\phi \phi}
$$

## Example 1: Stress around a circular pipe (tube)

Assume plane strain and plane stress, that the pipe is internally pressured with no applied stresses.
New boundary conditions are
$\sigma_{\mathrm{rr}}=-\mathrm{P}$ and $\sigma_{r \theta}=0$ at $r=r_{o}$
$\sigma_{r r}=\sigma_{r \theta}=0$ at $r=\infty$
Notice that $\Phi$ must be independent of $\theta$.
Now, let $r=e^{t}$
$\Phi=e^{m t}$ and substitute into the biharmonic.
A general solution is

$$
\Phi=A+B r^{2}+C \bullet \ln (r)+E r^{2} \boldsymbol{\bullet l n}(r)
$$

Check to see if $\Phi$ satisfies the biharmonic

$$
\frac{\partial^{2}}{\partial r^{2}}\left[\frac{\partial^{2} \Phi}{\partial r^{2}}+\frac{\partial^{2} \Phi}{\partial r^{2}}\right]+\frac{1}{r} \frac{\partial}{\partial r}\left[\frac{\partial^{2} \Phi}{\partial r^{2}}+\frac{1}{r} \frac{\partial \Phi}{\partial r}\right]=\frac{\partial^{4} \Phi}{\partial r^{4}}+\frac{1}{r} \frac{\partial^{3} \Phi}{\partial r^{3}}-\frac{1}{r^{2}} \frac{\partial^{2} \Phi}{\partial r^{2}}+\frac{1}{r^{3}} \frac{\partial \Phi}{\partial r}
$$

Now

$$
\begin{aligned}
& \frac{\partial \Phi}{\partial r}=\frac{C}{r}+2 B r+E r \cdot \ln (r)+E r \\
& \frac{\partial^{2} \Phi}{\partial r^{2}}=-\frac{C}{r^{2}}+2 B+E \cdot \ln (r)+3 E \\
& \frac{\partial^{3} \Phi}{\partial r^{3}}=\frac{2 C}{r^{3}}+2 B r+E r \cdot \ln (r)+E r \\
& \frac{\partial^{4} \Phi}{\partial r^{4}}=-\frac{6 C}{r^{4}}-\frac{2 E}{r^{2}}
\end{aligned}
$$

so...

$$
\begin{aligned}
& \begin{aligned}
& \nabla^{4} \Phi=\left[-\frac{6 C}{r^{4}}-\frac{2 E}{r^{2}}\right]+\frac{1}{r}\left[\frac{2 C}{r^{3}}+2 \frac{E}{r}\right]+\frac{1}{r^{2}}\left[-\frac{C}{r^{2}}+2 B+E \cdot \ln (r)+3 E\right] \\
&-\frac{1}{r^{3}}\left[\frac{C}{r}+2 B r+E r \cdot \ln (r)+E r\right]
\end{aligned} \\
& \text { and } \Rightarrow \nabla^{4} \Phi=0
\end{aligned}
$$

Now solve for the constants in the solution using the boundary conditions

$$
\begin{aligned}
\sigma_{r r}= & \frac{1}{r}\left(\frac{\partial \Phi}{\partial r}\right) \\
& =\frac{1}{r} \frac{\partial}{\partial r}\left(A+B r^{2}+C \ln (r)+E r^{2} \ln (r)\right)=2 B+\frac{C}{r^{2}}+2 E \ln (r)+E \\
\sigma_{\theta \theta}= & \frac{\partial^{2} \Phi}{\partial r^{2}}=2 B-\frac{C}{r^{2}}+E(3+2 \ln (r))
\end{aligned}
$$

As $r \rightarrow \infty, \sigma_{r r}, \sigma_{\theta \theta}=0$.
So E, B=0.

$$
\begin{aligned}
& \text { At } r=r_{0} \sigma_{r r}=-\mathrm{P} \\
& \left.\sigma_{r r}\right|_{r=r_{o}}=\frac{C}{r_{o}^{2}}=-P \\
& \sigma_{r r}=-P \frac{r^{2}}{r_{o}^{2}} \\
& \sigma_{\theta \theta}=-P \frac{r_{o}^{2}}{r^{2}} \\
& \sigma_{r \theta}=0
\end{aligned}
$$

.Remarks: Stress falls off as $r^{2}$, depends linearly on $P$ Hoop stresses are tensile.
If fractures occur they occur along planes of maximum tensile stress.

## Example 2: Stress and Strain Concentration around a Spherical Hole loaded by a pressure at infinity:

Strain about a spherical hole.
Suppose that a body loaded by pressure $\bar{\sigma}$ contains a spherical hole.
By symmetry we suppose that the displacements must be a function of $r$ only. Then guess that

$$
u_{r}=a r+\frac{b}{r^{2}}
$$

## Pore Compressibility:

Suppose we consider a spherical hole with an pressure $\bar{\sigma}$ applied at infinity.The material is elastic. From symmetry we suppose that the strain will be in the radial direction only, and that it will be proportional to the applied pressure:
$\varepsilon_{r r}=\frac{d R}{R}=C \cdot d \bar{\sigma} \quad$ where $C=\frac{1}{K_{0}} \frac{(1-v)}{2(1-2 v)}$
Then the change in pore volume is $d V_{p}=4 \pi R^{2} d R=3 V_{p} \frac{d R}{R}$
So the volumetric strain, $\varepsilon_{V}$ is
$\varepsilon_{V}=\frac{3(1-v)}{2 K_{0}(1-2 v)} d \bar{\sigma}$
and the pore stiffness is
$\frac{d V_{p}}{V_{p} d \bar{\sigma}}=\frac{3(1-v)}{2 K_{0}(1-2 v)}=\frac{1}{K_{\phi}}$

If the Poisson's ratio is 0.25 , then $1 / K_{\phi}$ is 2.25 ; as the solid material becomes incompressible the pore compressibility becomes large.


## Stress concentrations around pores, flaws, and inclusions:

Cylindrical Internal P
hole
" External P
" Uniaxial load, $\sigma$
Spherical Internal P
hole
"
External P
Uniaxial load, $\sigma$
Elliptical External P
hole
" Biaxial Load, S

