## Lecture #7

# Lagrange's Equations

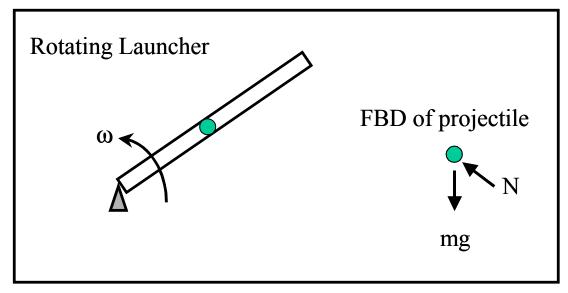
## Lagrange's Equations

## Joseph-Louis Lagrange 1736-1813

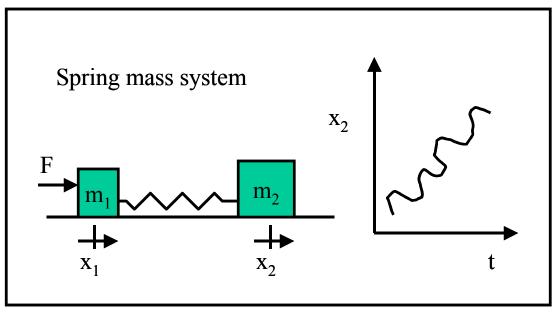
- <u>http://www-groups.dcs.st-and.ac.uk/~history/Mathematicians/Lagrange.html</u>
- Born in Italy, later lived in Berlin and Paris.
- Originally studied to be a lawyer
- Interest in math from reading Halley's 1693 work on algebra in optics
- "If I had been rich, I probably would not have devoted myself to mathematics."
- Contemporary of Euler, Bernoulli, Leibniz, D'Alembert, Laplace, Legendre (Newton 1643-1727)
- Contributions
  - o Calculus of variations
  - o Calculus of probabilities
  - Propagation of sound
  - Vibrating strings
  - o Integration of differential equations
  - o Orbits
  - o Number theory
  - ο...
- "... whatever this great man says, deserves the highest degree of consideration, but he is too abstract for youth" -- student at *Ecole Polytechnique*.

#### Why Lagrange (or why NOT Newton)

• Newton – Given motion, deduce forces



• Or given forces – solve for motion



Great for "simple systems"

What about "real" systems? Complexity increased by:

- Vectoral equations difficult to manage
- Constraints what holds the system together?
- No general procedures

#### Lagrange provides:

- Avoiding <u>some</u> constraints
- Equations presented in a standard form
- → Termed Analytic Mechanics
  - Originated by Leibnitz (1646-1716)
  - Motion (or equilibrium) is determined by <u>scalar</u> equations

#### **Big Picture**

- Use kinetic and potential energy to solve for the motion
- No need to solve for accelerations (KE is a velocity term)
- Do need to solve for **inertial** velocities

Let's start with the answer, and then explain how we get there.

#### **Define: Lagrangian Function**

• L = T - V (Kinetic – Potential energies)

## Lagrange's Equation

• For conservative systems

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = 0$$

• Results in the differential equations that describe the equations of motion of the system

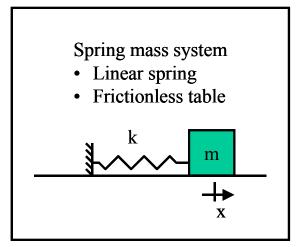
## Key point:

- Newton approach requires that you find accelerations in all 3 directions, equate F=ma, solve for the constraint forces, and then eliminate these to reduce the problem to "characteristic size"
- Lagrangian approach enables us to immediately reduce the problem to this "characteristic size" → we only have to solve for that many equations in the first place.

The ease of handling external constraints really differentiates the two approaches

## Simple Example

• Spring – mass system



• Lagrangian L = T - V

$$L = T - V = \frac{1}{2}m\dot{x}^2 - \frac{1}{2}kx^2$$

• Lagrange's Equation

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = 0$$

• Do the derivatives

$$\frac{\partial L}{\partial \dot{q}_i} = m\dot{x} \quad , \quad \frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) = m\ddot{x} \quad , \quad \frac{\partial L}{\partial q_i} = -kx$$

• Put it all together

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{q}_i} \right) - \frac{\partial L}{\partial q_i} = m\ddot{x} + kx = 0$$

Consider the MGR problem with the mass oscillating between the two springs. Only 1 degree of freedom of interest here so, take  $q_i = R$ 

$$\dot{r}_{M}^{I} = \begin{bmatrix} \dot{R} \\ 0 \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \omega \end{bmatrix}^{\times} \begin{bmatrix} R_{o} + R \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} \dot{R} \\ \omega(R_{o} + R) \\ 0 \end{bmatrix}$$
$$T = \frac{m}{2} (\dot{r}_{M}^{I})^{T} (\dot{r}_{M}^{I}) = \frac{m}{2} (\dot{R}^{2} + \omega^{2} (R_{o} + R)^{2})$$
$$V = 2 \frac{k}{2} R^{2}$$
$$L = T - V = \frac{m}{2} (\dot{R}^{2} + \omega^{2} (R_{o} + R)^{2}) - kR^{2}$$
$$\frac{d}{dt} (\frac{\partial L}{\partial \dot{R}}) = m\ddot{R}$$
$$\frac{\partial L}{\partial R} = m\omega^{2} (R_{o} + R) - 2kR$$

So the equations of motion are:  $m\ddot{R} - m\omega^2(R_o + R) + 2kR = 0$ or  $\ddot{R} + \left(\frac{2k}{m} - \omega^2\right)R = R_o\omega^2$  which is the same as on (3-4).

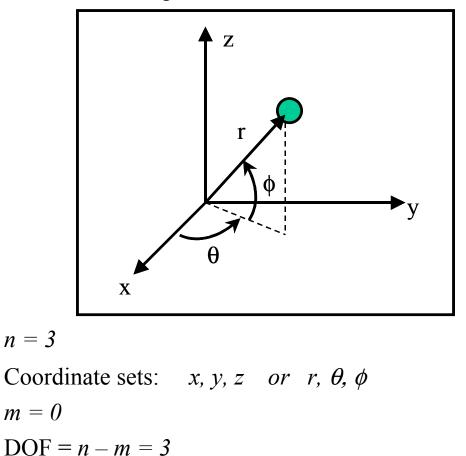
#### **Degrees of Freedom (DOF)**

- DOF = n m
  - $\circ$  *n* = number of coordinates
  - $\circ$  *m* = number of constraints

**Critical Point**: The number of DOF is a characteristic of the system and does <u>NOT</u> depend on the particular set of coordinates used to describe the configuration.

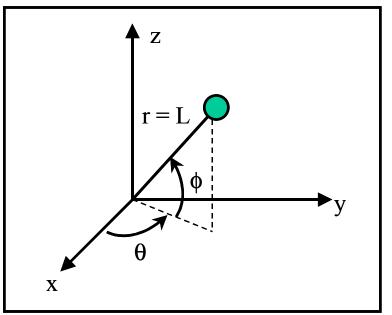
## Example 1

o Particle in space



#### Example 2





Cartesian Coordinates n = 3 (x, y, z) m = 1 ( $x^2 + y^2 + z^2 = R^2$ ) Spherical Coordinates  $n = 2 (\theta, \phi)$  m = 0DOF = 2

#### Example 3

DOF = 2

#### **Generalized Coordinates**

- No <u>specific</u> set of coordinates is required to analyze the system.
- Number of coordinates depends on the system, and not the set selected.
- <u>Any</u> set of parameters that are used to represent a system are called <u>generalized coordinates</u>.

## **Coordinate Transformation**

- Often find that the "best" set of generalized coordinates used to solve a problem may not provide the information needed for further analysis.
- Use a <u>coordinate transformation</u> to convert between sets of generalized coordinates.

**Example:** Work in polar coordinates, then transform to rectangular coordinates, e.g.

 $x = r \sin \theta \cos \phi$  $y = r \sin \theta \sin \phi$  $z = r \cos \theta$ 

#### **General Form of the Transformation**

Consider a system of N particles  $\rightarrow$  (Number of DOF = \_\_\_\_)

Let:

 $q_i$  be a set of generalized coordinates.

 $x_i$  be a set of Cartesian coordinates relative to an inertial frame

Transformation equations are:

$$x_{1} = f_{1}(q_{1}, q_{2}, q_{3}, \dots, q_{n}, t)$$
  

$$x_{2} = f_{2}(q_{1}, q_{2}, q_{3}, \dots, q_{n}, t)$$
  

$$\vdots$$
  

$$x_{n} = f_{n}(q_{1}, q_{2}, q_{3}, \dots, q_{n}, t)$$

Each set of coordinates can have equations of constraint (EOC)

- Let l = number of EOC for the set of  $x_i$
- Then DOF = n m = 3N l

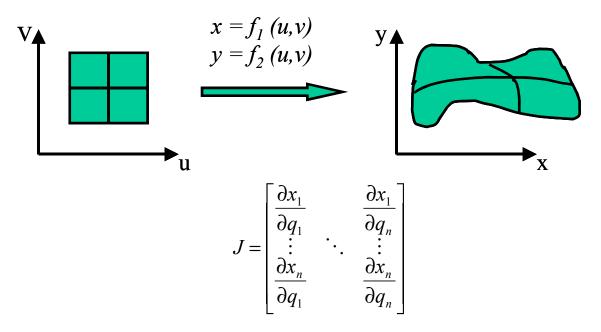
# **Recall**: Number of generalized coordinates required depends on the system, not the set selected.

#### Requirements for a coordinate transform

• Finite, single valued, continuous and differentiable

• Non-zero Jacobian 
$$J = \frac{\partial (x_1, x_2, x_3, \dots, x_n)}{\partial (q_1, q_2, q_3, \dots, q_n)}$$

• No singular points



**Example:** Cartesian to Polar transformation

 $\begin{array}{c} x = r\sin\theta\cos\phi \\ y = r\sin\theta\sin\phi \\ z = r\cos\theta \end{array} \rightarrow J = \begin{bmatrix} \sin\theta\cos\phi & r\cos\theta\cos\phi & -r\sin\theta\sin\phi \\ \sin\theta\sin\phi & r\cos\theta\sin\phi & r\sin\theta\cos\phi \\ \cos\theta & -r\sin\theta & 0 \end{bmatrix}$ 

$$|J| = \cos\theta [r^{2} \sin\theta \cos\theta \cos^{2}\phi + r^{2} \sin\theta \cos\theta \sin^{2}\phi]$$
$$+r \sin\theta [r \sin^{2}\theta \cos^{2}\phi + r \sin^{2}\theta \sin^{2}\phi]$$
$$|J| = r^{2} \sin\theta \neq 0 \text{ for } r \neq 0 \text{ and } \theta \neq 0 \pm n\pi$$

## **Constraints**

Existence of constraints complicates the solution of the problem.

- Can just eliminate the constraints
- Deal with them directly (Lagrange multipliers, more later).

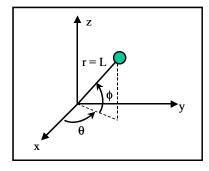
Holonomic Constraints can be expressed algebraically.

$$\phi_j(q_1, q_2, q_3, \dots, q_n, t) = 0, \ j = 1, 2, \dots, m$$

Properties of holonomic constraints

- Can always find a set of independent generalized coordinates
- Eliminate *m* coordinates to find *n m* independent generalized coordinates.

**Example:** Conical Pendulum



Cartesian Coordinates

n = 3 (x, y, z) m = 1 (x<sup>2</sup> + y<sup>2</sup> + z<sup>2</sup> = L<sup>2</sup>) DOF = 2 Spherical Coordinates  $n = (r, \theta, \phi)$ 

$$m = 1, r = L$$
  
DOF = 2

**Nonholonomic constraints** cannot be written in a closed-form (algebraic equation), but instead must be expressed in terms of the differentials of the coordinates (and possibly time)

$$\sum_{i=1}^{n} a_{ji} dq_i + a_{jt} dt = 0, \ j = 1, 2, \dots m$$
$$a_{ji} = \psi(q_1, q_2, q_3, \dots q_n, t)$$

• Constraints of this type are non-integrable and restrict the velocities of the system.

→ 
$$\sum_{i=1}^{n} a_{ji} \dot{q}_i + a_{jt} = 0, j = 1, 2, ...m$$

How determine if a differential equation is integrable and therefore holonomic?

• Integrable equations must be <u>exact</u>, i.e. they must satisfy the conditions: (*i*, *k* = 1,...,n)

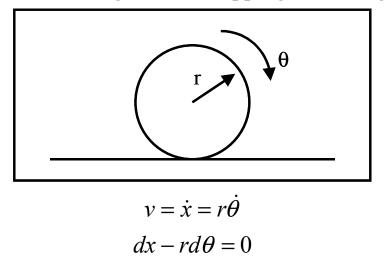
$$\frac{\partial a_{ji}}{\partial q_k} = \frac{\partial a_{jk}}{\partial q_i}$$
$$\frac{\partial a_{ji}}{\partial t} = \frac{\partial a_{jt}}{\partial q_i}$$

**Key point:** Nonholonomic constraints **do not** affect the number of DOF in a system.

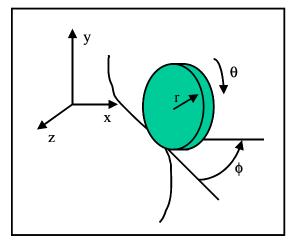
Special cases of holonomic and nonholonomic constraints

- **Scleronomic** No explicit dependence on *t* (time)
- **Rheonomic** Explicit dependence on *t*

**Example:** Wheel rolling without slipping in a straight line



**Example:** Wheel rolling without slipping on a curved path. Define  $\phi$  as angle between the tangent to the path and the x-axis.



 $\dot{x} = v \sin \phi = r \dot{\theta} \sin \phi$  $\dot{y} = v \cos \phi = r \dot{\theta} \cos \phi$  $dx - r \sin \phi \, d\theta = 0$  $dy - r \cos \phi \, d\theta = 0$ 

Have 2 differential equations of constraint, neither of which can be integrated without solving the entire problem.

➔ Constraints are nonholonomic

**Reason**? Can relate change in  $\theta$  to change in *x*, *y* for given  $\phi$ , but the absolute value of  $\theta$  depends on the path taken to get to that point (which is the "solution").

## Summary to Date

Why use Lagrange Formulation?

- 1. Scalar, not vector
- 2. Eliminate solving for constraint forces
- 3. Avoid finding accelerations

## **DOF – Degrees of Freedom**

- **DOF** = n m
- **n** is the number of coordinates
  - 3 for a particle
  - 6 for a rigid body
- **m** is the number of <u>holonomic</u> constraints

#### **Generalized Coordinates** $q_i$

- Term for any coordinate
- "Acquired skill" in applying Lagrange method is choosing a good set of generalized coordinates.

## **Coordinate Transform**

- Mapping between sets of coordinates
- Non-zero Jacobian